XFOIL Performance Validation for Medium-Scale Variable Pitch UAV Rotor Systems

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Abstract: This study focuses on experimentally validating the performance of XFOIL, a sophisticated software airfoil analysis tool used for approximating lift and drag coefficients. XFOIL output data was incorporated into a theoretical model simulating a variable pitch rotor system operating in a hovering state. The output of the Blade Element Momentum Theory (BEMT) rotor model is compared to thrust and power output performance data collected from a constructed rotor test bench and analysed in MATLAB. Using XFOIL as input, the BEMT rotor model was observed to yield good robust results when compared to experimental data, but demonstrated sensitivity to airfoil performance characteristics, laying the groundwork for future empirical validation. In comparing BEMT model performance, it was interesting to find that thrust performance remained within tolerance in contrast to an overprediction of rotor power output resulting from XFOIL drag at high blade pitch angles. Upon further interrogation by means of variable isolation, XFOIL demonstrated instability resulting from sensitivity to variability of model constraints. Modification of rotor geometry definitions or environmental constants beyond the test environment framework showed simulated systems may not necessarily behave reliably nor enhance output performance. This highlights the critical importance and utility of experimentation for understanding theoretical model behaviour or validating simulation output performance.

Additional keywords: AoA – Angle of Attack, XFOIL – Airfoil Analysis Application, BEMT – Blade Element Momentum Theory, UAV – Unmanned Aerial Vehicle, BEMT – Blade Element Momentum Theory, FSI – Fluid Structure Interaction, BLDC – Brushless Direct Current Motor

Nomenclature

- A_r Annulus Area $[m^2]$
- A Rotor Area $[m^2]$
- a_B Blade Mach Number
- B Chord Length [m]
- C_D Drag Coefficient
- C_L Lift Coefficient
- C_P Power Coefficient
- C_T Thrust Coefficient
- c_{ISASL} Speed Of Sound Sea Level [m/s]
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- GTW Gross Take-off Weight [kg]
- I_b Mass Inertia [$kg.m^2$]
- *I_{sm}* Sample Mean Current [*A*]
- I_m Mean Current [A]
- I_s Sample Current [A]
- l_b Blade Length [m]
- L Energy $[Kg.m^2/s]$
- N_r Rotor Speed / Head Speed [r/s]
- n_i Blade Section Increment [n]
- n_p Pitch Sample Rate / Resolution [n]
- n_I Current Sample Rate / Resolution [n]
- n_{raw} Raw Sample Rate / Resolution [n]
- *N_b* Number Of Blades
- P_s Sample Power Output [W]
- $\vec{P_{poly}}$ Polynomial Power Output [W]
- *Re* Reynolds Number
- *r_i* Radial Position Increment
- T_{sm} Sample Mean Thrust [Kg]
- T_m Mean Thrust [N]
- T_s Sample Thrust [N]
- T_{poly} Thrust Polynomial [N]
- V_{ave} Average Voltage [V]
- V_m Mean Voltage [V]
- V_s Sample Voltage [V]
- v_{kin} Kinematic Viscosity [N. s/m²]
- α Lift Slope Constant
- θ_{ix} Increment Pitch Angle, (AoA) [Deg]
- τ_{net} Net Torque [N.m]
- ω_r , ω_b Rotational Speed, Tip Speed [rad/s]
- λ Induced Velocity [m/s]
- σ Rotor Solidity $[N/m^2]$

1 Introduction

The UAV market is expected to triple in size by 2027, and the significant investment forces generated by large corporations have accelerated technological development efforts of quad-rotorcraft platforms for light transport, agricultural, and surveillance applications. [1] With advancing electronics, aerodynamics, and materials science, sophisticated rotorcraft technologies will seamlessly integrate into everyday life, becoming nearly imperceptible. [2] Given their success, these platforms continue facing scalability challenges attributed to factors such as energy density constraints and propulsion efficiency. [3] The optimization challenge lies in expanding mission profiles while balancing performance expectations through trade-offs in endurance, payload capacity, cost, and complexity. [4-6]

Focusing on propulsion efficiency challenges highlighted earlier – For quadrotors employing fixed-pitch propulsion systems, manoeuvring and stabilization is achieved by altering the thrust balance of opposing rotors through rapid speed modulation, requiring greater effort from motors to

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overcome resulting inertial forces produced by rotors. BEMT theory and other works [7-9] show that rotors incur efficiency losses from work required to maintain altitude at low flight speeds. Fixed-pitch propulsion systems used in modern UAV's [10] are optimized for specific [7-9,11] Combined, the technical challenges presented earlier are especially emphasized in medium scale quad-rotorcraft (GTW>10 kg) considering how payload and endurance are intwined. [12-15]

If the technical challenges shown in work examining the adaption of variable pitch propulsion technology for quadrotors could be overcome [16-20], the benefits of higher endurance and payload capacities resulting from optimizing rotor efficiency could have significant commercial implications.

While software-based modelling and simulation strategies applied to the mentioned problems could provide high-resolution insights, aircraft are sensitive to aerodynamic performance characteristics, often requiring empirical validation to ensure reliable performance. [21] Consequently, this work will focus on empirically evaluating the scalability performance in terms of rotor geometry of a variable pitch rotor system scaled for a medium size quadrotor platform using MATLAB and XFOIL.

Developing a MATLAB-based rotary propulsion system is dependent on combining momentum principles with elemental airfoil flow theory [8,22]. Airfoil performance characterized in terms of lift and drag coefficients is traditionally evaluated empirically from measurements obtained in wind tunnel testing [23]. Due to limited cost and access to test equipment, airfoil flow behaviour can also be simulated using software tools such as Ansys Fluent or Ansys CFX which offer sequential (One-Way Coupling) or parallel (Two-Way Coupling) analysis schemes depending on the significance of the FSI (fluid structure interaction) effect [24,25]. In this case, the BEMT model will rely on a wellestablished wind tunnel emulator XFOIL known for its ease of use, robustness, and computational efficiency. In this work, XFOIL is used to simulate the airfoil lift (C_L) and drag (C_D) coefficients using code developed using potential flow panel and integral boundary layer formulation methods to predict flow separation. Since the solver assumes flow to be two-dimensional, inviscid, and incompressible it may not accurately represent real-world scenarios with threedimensional or compressible effects. [21.24.26]

It was observed that the simulation of the theoretical rotor model correlated with experimental results.

Theoretical Formulation 2

The MATLAB-based model used to simulate variable pitch rotor system (see §4) incorporates well established Blade Element Momentum Theory (BEMT) referenced from Bramwell, et al. [8] and Johnson [7]. To simulate maximum power demand scenario, the rotor is assumed to be in a hovering condition using rigid, untwisted blades.

Coefficients for thrust (1) and power (2), C_P and C_T are obtained by integrating over the blade length from radius $R_o \rightarrow R_{max}$ at a given angle of attack θ_i form lift and drag coefficients C_D , C_L obtained from XFOIL.

Rotor power output is defined from coefficients of airfoil profile drag dC_{Po} (3) to overcome drag acting on the blade, and induced power dC_{Pi} (4) to generate lift. Since velocity ω_r (8) increases with radius station r_i , dC_{Po} and dC_{Pi} are integrated over the blade span at blade angle increment θ_i , with $n_i = 100$ blade sections.

$$C_T = \int_{R_0}^{Rmax} \frac{1}{2} \sigma C_L r_i^2 dr \tag{1}$$

$$C_P = dC_{Pi} + dC_{Po} \tag{2}$$

$$dC_{Po} = \int_{R_o}^{Rmax} \frac{1}{2} \sigma C_D r_i^3 dr \tag{3}$$

$$dC_{Pi} = \int_{R_o}^{Rmax} \lambda dC_T \tag{4}$$

$$dC_T = \frac{1}{2}\sigma a \int (\theta_i r_i^2 - \lambda r_i) dr$$
⁽⁵⁾

From the equations above, and taking air density as $\rho =$ $1.224 kg/m^3$ – rotor thrust T_r (6), and power consumption P_r (7), is then calculated by:

$$T_r = C_T \rho A_r (\omega_r R_{max})^2 \tag{6}$$

$$P_r = C_P \rho A_r (\omega_r R_{max})^3 \tag{7}$$

$$\omega_r = \frac{2\pi N_r}{60} \tag{8}$$

2.1 Rotor Solidity, Inflow Velocity, Lift Slope and Blade Area

Rotor solidity σ (9) defines thrust force per unit area of the annulus and is the function of the number of blades N_h , chord length c, and rotor tip radius R_{max} .

$$\sigma = \frac{N_b c}{\pi R_{max}} \tag{9}$$

Combining blade element and momentum theory [22], for a rotor in a hovering condition with non-uniform inflow, induced velocity λ (10) can be calculated:

$$\lambda = \frac{\sigma a}{16} \left[\sqrt{1 + \frac{32}{\sigma a} \theta_i r_i - 1} \right]$$
(10)

Airfoil lift slope a (11) is defined from the ratio of lift coefficient C_L (output from XFOIL) to incidence angle α .

$$a = \frac{c_L}{\alpha} \quad (\alpha = 5.7) \tag{11}$$

In accordance with momentum conservation laws [27], a net torque, τ_{net} produces a rotating motion in a body, where changes in τ_{net} causes an angular acceleration (12). In this study, we assume a constant rotor head speed (N_r) therefore angular velocity ω_b and τ_{net} are constant, conserving momentum. Thus, balanced torques result in no angular acceleration, therefore $\tau_{net} = 0$ (13).

$$L = I_b \omega_b \left(Kg. m^2 / s \right) \tag{12}$$

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$$\tau_{net} = \frac{\Delta L}{\Delta t} = 0 \tag{13}$$

Any net torque τ_{net} (13) transmitted during operation is therefore equal to mechanical friction of the drive-train and hub of the Brushless Direct Current (BLDC) Motor, as well as aerodynamic lift and drag forces acting on the rotor blades. (See §3.1)

3 Rotor Test Bench Design

A rotor test bench (figure 1, below) was constructed to serve as the empirical framework for validation of XFOIL and BEMT models outlined in §4.



Figure 1 Rotor Test Bench

Various sets of rotor blades (figure 2, below) with specifications listed in table 2 were tested at specific rotational speeds (N_r) and pitch angles (θ_{ix}). Sample data collection and processing are further detailed in §3.5.



Figure 2 Rotor Blade Sets

3.1 Technical Description & Specifications

In reference to figure 1 above – an APD 200F3 electronic speed controller (ESC) powers a V10L T-Motor BLDC coupled to 1:1 bevel gearbox. Collective pitch is modulated by a servo, and current and voltage are detected using an MSC1500 hall effect sensor and voltage divider. IR sensors monitor rotor head speed, and a 50kg load cell connected to a HX711 amplifier measures thrust. The Arduino Nano logs data collected from the servo and sensors via a serial monitor. Table 1 provides an overview of the rotor test bench specifications.

Table 1	Test Bench S	Specifications

Description	Min	Max
Power Output (kW)	0.1	8
Motor Speed (r/s)	0	6000
Rated Head Speed (r/s)		3000
Blade Pitch Angle (Deg.)	- 5.0 (±0.5)	+35.0 (±0.5)

Thrust Capacity (Kg)	50 (±0.01)		
**Construction	Stainless Steel & Aluminium		
Gear Ratio	1:1		

**Construction is defined as mechanical components, excluding electronics

Table 2 Rotor Blade Specifications

В	R _{max} (mm)	R _{root} (mm)	С (mm)	<i>В</i> (mm)	b (mm)	NACA	M_b (g)	$R_g(\text{mm})$
B03	325.0	65.0	34.7	4.98	9.0	0014	26.4	167.5
B04	360.0	69.0	34.0	4.50	10.0	0014	31.4	176.0
B05	430.0	90.0	42.0	5.75	11.0	0014	72.6	207.5

**B – Blade Set, R_{max} – Blade length, R_{root} – Annulus Root Radius, C – Chord Length, B – Chord Thickness, b – Distance from leading edge to thickness B, M_b – Mass of a single blade, R_g – Mass Centroid about the X-Axis

3.2 Test Bench Working Principle

As illustrated in figure 3, the load cell is positioned such thrust force T (3) is transmitted to the load cell via reactionary force F (1) along the Z-axis. Bearing guides sliding on rigidly mounted guides isolate residual reactionary forces F_r (2) or moments M (4) about X|Y axes to prevent torsional forces acting on the load cell.

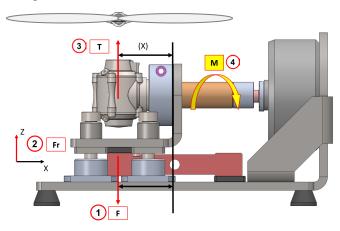


Figure 3 Rotor Test Bench (See Appendix D)

3.3 Experimentation

Three blade sets were tested at constant speed ($N_r = 1000 - 2500 rpm$) with a collective pitch of $\theta_{ix} = 0^o - 14^o$. Sample data was processed and conditioned in MATLAB before being compared to theoretical performance results. The sample collection and processing procedure is summarized below.

- 1. Set rotor to specific speed with $\theta_{ix} = 0^{\circ}$,
- 2. TARE load cell to zero thrust readout,
- 3. Collect $n_I \ge 10$ samples for $(V_{sm}, I_{sm}, T_{sm}, \theta_{ix})$ at pitch increment θ_{ix} ,
- 4. Increase pitch $\theta_{ix} + 0.25^{\circ}$ and repeat for entire pitch range mentioned above.

3.4 Raw Sample Collection – Arduino Nano

Voltage (V_s) and current (I_s) are measured at pitch increment θ_{ix} and accumulated with a sample rate of $n_{raw} = 100$. Mean values V_{sm} and I_{sm} are calculated and output on the serial monitor from (14) and (15):

$$V_{sm} = \frac{\Sigma V_s}{n_{raw}} \tag{14}$$

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$$I_{sm} = \frac{\sum I_{sm}}{n_{raw}} \tag{15}$$

Raw thrust (T_s) samples are collected and averaged over n_{raw} from (16):

$$T_{sm} = \frac{T_s}{n_{raw}} \tag{16}$$

The pitch angle (θ_i) measurement range is determined by manual calibration using a dial gauge with setpoints θ_{max} , θ_{min} corresponding to the pulse width modulated (PWM) signal output range γ_{max} , γ_{min} .

Pitch angle θ_{ix} (17) can then be determined by converting current servo position (γ_i) to degrees using linear interpolation. (Table 1)

$$\theta_{ix} = \theta_{min} + \left(\frac{\gamma_i - \gamma_{min}}{\gamma_{max} - \gamma_{min}}\right) (\theta_{max} - \theta_{min})$$
(17)

3.5 Sample Pre-Processing & Signal Conditioning

MATLAB is used to process raw sample data $(T_{sm}, I_{sm}, V_{sm,}, \theta_{ix})$ to calculate the indicated power consumption (20) & thrust (21) produced by the rotor system. (Figure 4)

Over the duration of a test run for a single set of blades, voltage drop was observed to be consistently small ($V_{ave} = 2 - 2.5\%$). Mean voltage, V_m (18) is calculated by averaging accumulated raw samples V_{sm} for duration of the test run (n_p) at a constant speed of N_r using:

$$V_m = \frac{V_{sm}}{n_p}$$
, where n_p = sample range (18)

As blade pitch (θ_{ix}) increases, more work is done by the rotor to move air, consequently leading to higher current demands from the motor to maintain speed. Mean current (I_m) is calculated at each pitch increment (θ_{ix}) from:

$$I_m = \frac{l_{sm}}{n_l}$$
 at θ_{ix} where $n_l \ge 10$ (19)

Where n_I is the sample range collected at increment θ_{ix} .

Using the mean voltage (18) and current (19), rotor power output (P_s) can then be calculated by applying Ohm's Power Law:

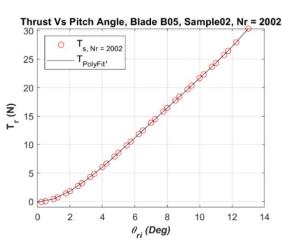
$$P_s = V_m I_m, \text{ at } \theta_{ix} \tag{20}$$

Using the same sample range for (19), Rotor thrust T_r is calculated by converting mean thrust (T_{sm}) samples accumulated at increment θ_{ix} into Newtons $(g = 9.810 \text{ m/s}^2)$ and then averaging over n_I (21):

$$T_r = \frac{(gT_{sm})}{n_I}$$
, at θ_{ix} where $n_I \ge 10$ (21)

Due to the 10bit resolution limitation of the *ATmega 328* microcontroller used in the Arduino Nano, the indicated pitch angle resolution was limited to $\theta_{ix} \pm 0.25^{\circ}$. Discrete outputs of θ_{ix} that didn't correspond to integers ($\theta_{ix} \notin Z$) were approximated in MATLAB by applying a 3rd degree polynomial curve fitting method [28] as it was challenging to perform a direct comparison to theoretical approximations described in §2. The polynomial form is represented as $\theta_i(x) = p(x) = \sum_{k=0}^n p_k x^k$ (22). Results for thrust and power consumption (T_{Poly}, P_{Poly}) were obtained as shown in figure 4.

$$p(x) = p_n x^n + p_{n-1} x^{n-1} + \dots + p_2 x^2 + p_1 + p_0$$
(22)



Power Vs Pitch Angle, Blade B05, Sample02, Nr = 2002

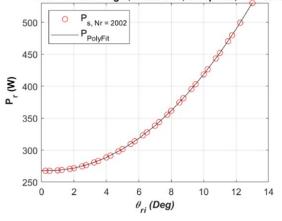


Figure 4 Sample Pre-Processing, B05

4 Rotor Modelling & Analysis

Analysing the theoretical performance characteristics of rotors necessitated the development of a two-step method. Matching the test environment in which Re, $M_B = const$, performs a polar analysis of the airfoil, accumulating lift and drag coefficients at pitch angles θ_{ix} following the workflow illustrated in figure 7, Appendix A. MATLAB [29] then evaluates the rotor performance for thrust (T_r) and power output (P_r) of the BEMT rotor model (see figure 9, Appendix B and figure 10, Appendix C) using data defined in table 2, and output from XFOIL.

Firstly, XFOIL is configured to perform an airfoil analysis to evaluate lift and drag properties. This analysis method is also useful for interrogating airfoil performance characteristics such as stall behaviour and for optimising aerodynamic efficiency. The following respective are commands required to configure XFOIL for polar accumulation of lift and drag coefficients, C_l , C_d . Refer to user documentation for definitions [26]:

- 1. NACA Airfoil number (table 2)
- 2. OPER: Operation selection,
- 3. ITER: Number iterations = 100,
- 4. VISC Set viscous analysis mode to active,

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- 5. RE Input Reynolds Number (Re), see below,
- 6. MACH Set Critical Mach number (a_B) , see below
- 7. SEQP Select sequential polar analysis mode,
- 8. PACC Set to polar accumulation active,
- 9. ASEQ Set pitch angle range and increment $(0^{o} 13^{o}, \theta_{ix} + 1.0^{o})$.

Reynolds number is determined from $Re = V_r B/v_{kin}$, where $v_{kin} = 1.5111E^{-5}$ N. s/m² (air properties at ISASL), tip velocity $V_r = \omega_r R_{max}$, and chord length *B* (table 2). Mach number is calculated from $a_B = V_r/c_{ISASL}$ where the speed of sound for air at sea level is $c_{ISASL} = 343$ m/s.

Rotor thrust and power output is determined for each pitch increment of $\theta_{ix} = 1.0^{\circ}$ using a MATLAB-based rotor model based on BEMT (see §2). Code originally developed by Bell [29] was extensively modified to integrate XFOIL outputs and optimized to automatically accumulate sequential calculations as illustrated in figure 5. The analysis method follows the respective steps below according to the workflow illustrated in figure 8, Appendix A.

- 1. Import lift and drag coefficients from XFOIL text file,
- 2. Define blade specifications (table 2),
 - Number of blades N_b ,
 - Rotor Max Radius R_{max} (m),
 - Rotor root radius R_o (m),
 - Chord length B (m),
- 3. Air density, $\rho_{ISASL} = 1.225 \text{ kg/m}^3$,
- 4. Pitch angle range, $\theta_{ix} = 0^o 13^o$ (deg.),
- 5. Rotor Speed N_r (r/s).

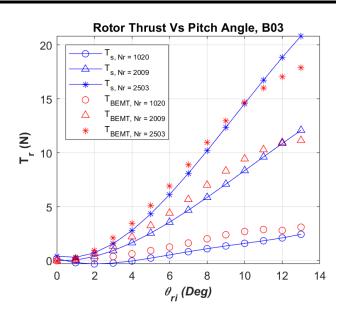
5 Results & Findings

Comparisons discussed in section 5 use blade set B03 (figure 5) for illustrative purposes, comparisons for B04 & B05 are shown in figure 9 (Appendix B) and figure 10 (Appendix C).

5.1 Thrust Comparison Overview

Theoretical approximations for thrust output by BEMT and XFOIL closely matched experimental data (T_{BEMT}, T_{sample}) at speeds ($N_r = 1000 \rightarrow 2500$ rpm) and rotor pitch angles ($\theta_i = 0^o \rightarrow 14^o$) for all rotor blades tested (table 2).

As shown in figure 5, the consistency of the MATLABbased rotor model's performance, particularly in terms of thrust and power output, is noteworthy when considering the variations in geometry and speeds that were tested. This consistency implies that the simulated environment in MATLAB closely aligns with the conditions observed in the experimental tests. In other words, the results obtained from the MATLAB simulation closely match the outcomes observed during physical testing. This alignment suggests that the simulated and real-world environments share a strong resemblance, thereby indicating the reliability and accuracy of the MATLAB model in predicting the thrust performance of the rotor system.



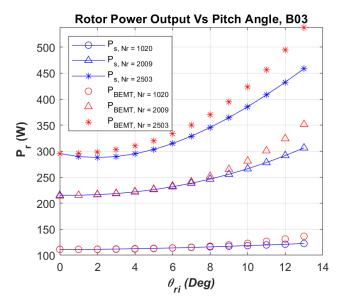


Figure 5 Sample 01, B03

5.2 Power Output Comparison Overview

When operating rotor blades at speeds $N_r \approx 1000 - 2000$ rpm, theoretical power consumption estimates (P_{BEMT}) were observed to follow experimental test results (P_{sample}), as $P_{sample} \approx P_{BEMT}$ within a pitch range of $\theta_i = 0^\circ \rightarrow 7^\circ$. At larger pitch angles ($\theta_i \approx 7^\circ \rightarrow 13^\circ$), theoretical power outputs are overestimated ($P_{BEMT} > P_{sample}$) for all rotor blades tested at all speed ranges. To try and establish deviations observed for power output mentioned prior, a further investigation was conducted as is discussed in §5.3.

5.3 Investigation of XFOIL Instability

Applying configuration settings and variables (see §3.3), XFOIL outputs C_l , C_d for each increment of θ_i after which the BEMT rotor model (illustrated in figure 8, Appendix A) outputs performance data according to the blade set as defined in table 2. All rotor experiments were performed in an ideal environment matching the model geometry

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definition and processed in MATLAB to remove sensor noise.

From momentum theory it is shown that outputs T_r , P_r are dependent on constants ρ , A_r , ω_r , R_{max} relating to thrust and power coefficients (C_T , C_P), which in turn rely on C_l and C_d output by XFOIL. [7,8] Given the rotor scale, model performance is highly dependent on the accuracy of C_l , C_d , especially for low Reynolds numbers in which experiments were conducted. [21,24]

From comparisons (figure 5), it observed that thrust (T_{BEMT}) behaviour remained stable $(e_{Tr} \approx 10 - 12\%)$ indicating that BEMT rotor model can be used reliably with XFOIL to analyse thrust performance scalability of medium scale rotor systems. In contrast, P_r was consistently overestimated, and so a deeper analysis of the rotor model and XFOIL configuration was conducted.

The BEMT power term, P_{BEMT} (7) is dependent on C_P (2) – the sum of lift induced drag dC_{Pi} (3) and parasitic drag dC_{Po} (4). Adjustment to rotor definitions and environmental constants (R_{max} , B, v_{kin}) when simulated according to tests (where $N_r \approx 1000 - 2500 \ rpm$ and $\theta_i \approx 0^o \rightarrow 13^o$) did not improve prediction for P_{BEMT} – but it was found that the contribution of dC_{Pi} remained stable as C_l output by XFOIL remained almost unchanged.

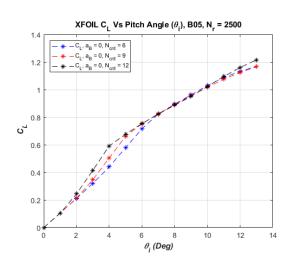
Further expanding on findings – the increase in P_{BEMT} at higher pitch angles ($\theta_i \approx 7^o \rightarrow 13^o$) was found to be resulting from the contribution of dC_{Po} which are dependent on C_d output from XFOIL.

To investigate C_d variance, the analysis described above was repeated for B05 (Table 2) with rotor speed set to $N_r =$ 2500 rpm focusing on isolating the air quality factor N_{crit} (see §4). Output results are plotted for C_L and C_D as shown in figure 6.

Before discussing results, it's important to note that the $N_{crit} = 3 \rightarrow 14$ (range) parameter in XFOIL defines air disturbance level in which the airfoil operates in from poor to ideal. In this work (figure 5), $N_{crit} = 9$ is used to assume average air turbulence quality. Authors of XFOIL note that calculating dynamic pressure over the airfoil may result in convergence errors since flow is dominated by viscous effects when $Re < 1x10^6$ and compressibility effects can be assumed to be negligible. [26] At $N_r = 2500 rpm$, rotor produces $Re = 3.13x10^5$ which is below the aforementioned Re threshold, thus MACH is prescribed as $a_B = 0$.

As an example (figure 6), C_d correlates to increasing values for $N_{crit} = 6 \rightarrow 12$ when $a_B = 0$, especially when the blade enters the stall region ($\theta_i \approx 7^o \rightarrow 13^o$). As shown, the N_{crit} has no significant effect on reducing C_d which contributes to dC_{Po} leading to overestimating P_{BEMT} .

It was intriguing to discover that XFOIL appeared to stabilize at higher Reynolds numbers ($Re > 1x10^6$) with minimal improvements for C_d when augmenting model parameters R_{max} , B, rotor speeds (N_r), or environmental constants (v_{kin}). However, no quantitative comparisons or conclusions could be made because the model definition no longer reflected the actual test scenario.



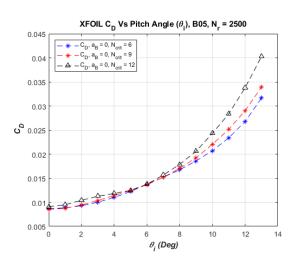


Figure 6 XFOIL Comparisons for C_L and C_D

6 Conclusion

The non-linear behaviour exhibited by aerodynamic systems is inherently challenging to approximate, highlighting the importance of experimental validation for simulated performance. As comparisons (figures 5, 9, 10) for all rotor geometry variants demonstrate, the BEMT rotor model accurately emulated the test environment indicating that rotor geometries, speeds and environmental constraints are comprehensively defined.

Thrust (T_{BEMT}) approximations are consistently predicted within a tolerance of $e_{Tr} \approx 10 - 12\%$ at varying speed ranges $(N_r = 1000 \rightarrow 2500 \text{ rpm})$. Closing the error gap will require the inclusion of more advanced theoretical assumptions with regards to rotor aerodynamics such (tip losses etc.), environmental variabilities and more sensitive test equipment.

In contrast to thrust, rotor power output (P_{BEMT}) was overestimated showing BEMT model performance remains sensitive to the accuracy of airfoil characteristics as was found in similar work. [21,24] Interrogation of P_{BEMT} revealed XFOIL instability to predict C_d and may be related to low Reynolds numbers (*Re*) which possibly lie beyond what the software could resolve. Artificially modifying model definitions to increase *Re* showed some improvement

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to predict C_d , but model definitions no longer mirrored the test environment.

Reviewing the comparisons above as depicted in figures 5, 8 & 9, it is important to emphasize that the BEMT model demonstrated consistent performance, but exhibits sensitivity to the precision of environmental factors, model constraints, and airfoil properties output from XFOIL. Variable isolation efforts of model definitions also revealed evidence of a complex interdependence between multiple parameters which influence the contribution of drag C_d to dC_{Po} , indicating that care should be taken when approximating airfoil performance characteristics.

Finally, this work demonstrated that the theory and methodology outlined in this work has practical utility and can be directly applied during early design parametrization efforts to establish accurate thrust and power estimations. This investigation has inspired efforts for future such as

- Refining the BEMT rotor model to enhance the precision of thrust and power predictions,
- Analysing complex parameter interdependencies that influence drag contribution to better understand their impact on the performance of the BEMT rotor model.
- Validation with a wider variety of rotor geometries and operating conditions to further validate the performance and applicability of the BEMT rotor model in a variety of scenarios.

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7 Appendix A

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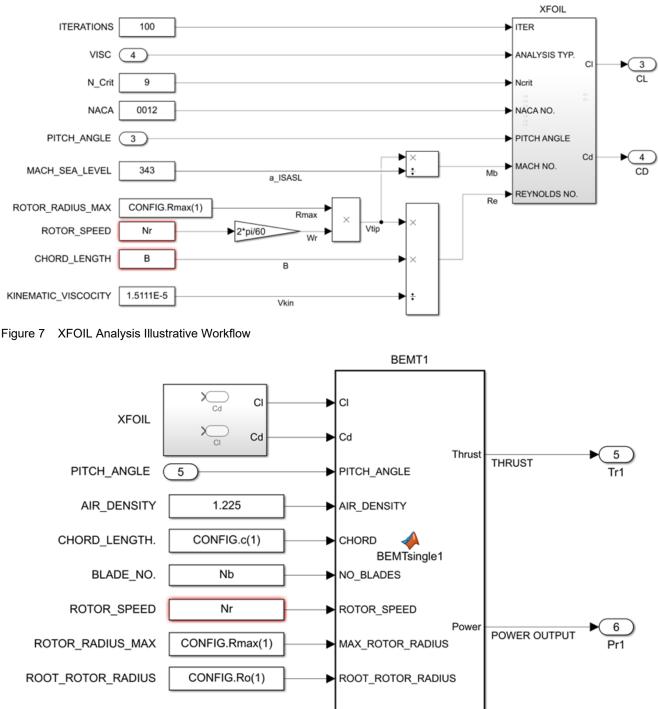


Figure 8 MATLAB Modelling Illustrative Workflow

8 Appendix B

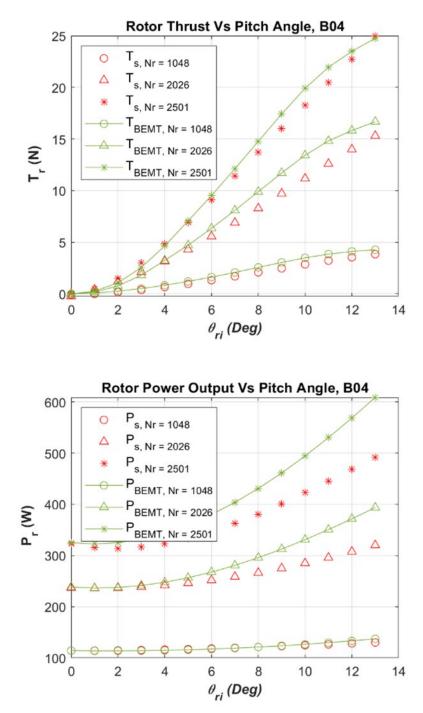


Figure 9 Sample 03, B05

9 Appendix C

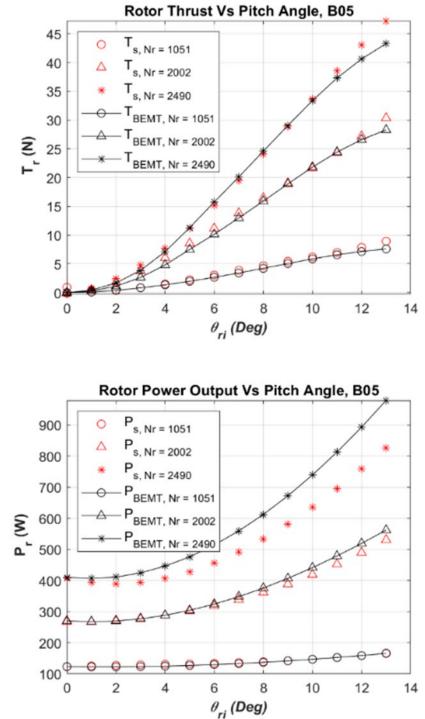


Figure 10 Sample 02, B04

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10 Appendix D

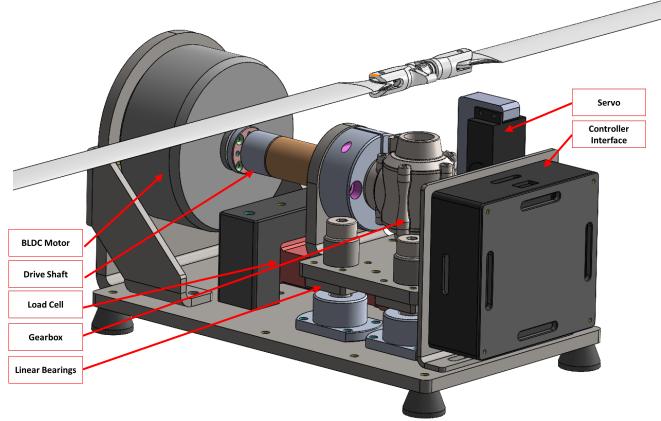


Figure 11 Rotor Bench Isometric View