Influence of fire-ignition methods and stove ventilation rates on gaseous and particle emissions from residential coal braziers

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Abstract
Despite extensive electrification of low-income residential areas on the South African Highveld, extensive use is still made of wood and coal as domestic fuels, particularly for winter space heating. In informal settlements characterised by poverty and lacking electrification, coal is combusted in non-standardised, inefficient and polluting metal braziers, colloquially known as imbaulas. Copious emissions from domestic coal fires result in elevated household and ambient air pollution levels well above national air quality limits. Despite the severity of this pollution as a public health issue, emissions data from residential coal-burning braziers are still scarce. Consequently, there is a need to evaluate the emission characteristics of these devices. In this paper, we report on controlled combustion experiments carried out to investigate systematically influences of fire-ignition methods and stove ventilation rates on gaseous and condensed matter (smoke) emissions from informal residential coal combustion braziers. Two methods of stove ignition—conventional bottom-lit updraft (BLUD) and the top-lit updraft (TLUD) (colloquially known as the Basa njengo Magogo) were investigated. Emission factors (EFs) were found to be dependent on fire ignition method and stove ventilation rates. The top-lit ignition method reduces PM10/PM2.5 by 76% to 80%

compared to the BLUD method. Carbon monoxide emissions do not change significantly with the ignition method. Pollutant emissions from normal combustion in high ventilation conditions were low compared to pollutants emitted when an oxygen deficient atmosphere was created under low ventilation conditions. High stove ventilation rates resulted in a 50% reduction in PM10/PM2.5 emissions compared to the low ventilation rates. Emissions of gaseous and particulate matter from incomplete combustion can be minimised by design optimisation of the braziers. Ideally, the emissions of condensed matter particles (which form the bulk of emitted particles in residential coal fires) can be reduced through good mixing of emitted gases and air, and by ensuring a long residence time in the high temperature oxygen rich post combustion zone, to allow for complete oxidation. This study is significant in that it presents the first systematic and comprehensive study of factors affecting emissions from coal braziers.

Keywords: household air pollution, residential coal combustion, emission factors, braziers, imbaula, improved stoves.

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1. Introduction

Household air pollution, primarily from cooking and heating stoves, has been identified as a major global cause of morbidity and mortality from non-communicable diseases (Lim *et al*. 2012). Potentially harmful combustion emissions include carbon monoxide (CO), particulate matter (PM), organic carbon (OC), elemental carbon (EC), and polycyclic aromatic hydrocarbons (PAHs) (Shen *et al*., 2013). Specifically, fine and ultrafine particulate matter emissions from coal combustion are receiving significant attention from regulatory authorities and environmental scientists, due to their adverse health effects (Makonese *et al*., 2014; Bai *et al*., 2011; Teramae & Takarada, 2009; Nelson, 2007; Yi *et al*., 2008; Bockhorn, 2000). Hence, fine particulate formation mechanisms and gas/particle partitioning, and influencing factors need to be studied in detail to develop novel combustion technologies that are less polluting (Bai *et al*., 2011).

Starting in the early 1980's, the large-scale electrification of the apartheid government-designated Black residential areas was regarded, at the time, as the solution to the pervasive air pollution problems associated with the use of coal and wood as domestic heating and cooking fuels. While access to electricity was welcomed as a contribution to improved quality of life and increased modernity, the hoped for coincident improvement in ambient air quality did not take place due to several reasons. The dominant reason was economic – for heating, coal is more economical than electricity, by at least a factor of two (Kimemia & Annegarn, 2011a).

In South Africa, coal fuel will continue to be a major industrial and domestic source of energy for several decades. This energy carrier will continue to be used as a primary source of energy for generation of electricity, industrialisation and for the enhancement of standard of living for the increasing population (Finkelman *et al*., 2002; IRP, 2013). In burgeoning non-electrified informal settlements on the interior plateau of South Africa, commonly known as the Highveld, continued combustion of wood and coal in hand-fabricated braziers provides the primary domestic energy source. These braziers (colloquially known as *imbaulas*) are typically fabricated by punching holes in the sides of discarded metal paint drums and fitting a crude wire grid (Makonese, 2011; Kimemia *et al*., 2011; Le Roux, *et al*., 2009; Standish *et al*., 2007). These devices are used extensively for space heating during the winter (May–August) (Makonese, 2011; Kimemia *et al*., 2011b; Annegarn & Sithole, 1999). In the absence of any design optimisation, these braziers generate copious quantities of dense white smoke, composed primarily of condensed semi-volatile hydrocarbons from the pyrolysis and incomplete combustion of coal and wood. Source-apportionment studies have identified residential coal burning as the greatest single source of airborne aerosols among other pollutants in South Africa's coal-burning townships (Mdluli, 2007; Scorgie *et al*., 2001; Annegarn *et al*., 1998). The levels of combustion particles are particularly high on cold, clear nights with calm winds, especially when low-lying inversion layers reduce vertical exchange of air (Bølling *et al*., 2012; Namork *et al*., 2004).

There are, to date, a dearth of reports on emission factors (EFs) from residential coal combustion, leading to uncertainties and bias in most emission inventories and predictive models (Shen *et al*., 2013). A compilation of EFs for residential coal combustion varied by a factor ten (Scorgie *et al*., 2004). The variability arises in part from the variety of parameters that influence combustion. For example, combustion technology and operational practice of appliances have a major influence on the physico-chemical properties of the emitted particles (Uski, 2014; Bølling *et al*., 2012; Tissari *et al*., 2008). Emission factors reported in the literature vary due to differences in fuel properties (*e.g.* moisture and volatile matter content), stove design, fire ignition methods (BLUD versus TLUD), fire management and ventilation (*e.g.* air supply amount and fuel-air mixing conditions) and even experimental methods (*e.g.* laboratory chamber, simulated stove/open burning, and field measurement) (Shen *et al*., 2013; Chen *et al*., 2012; Jetter *et al*., 2012; Johnson *et al*., 2010; Jetter & Kariher, 2009).

The aim of this paper is to report on a systematic study of some of the factors that influence emission factors from packed-bed residential coal-burning devices, namely influences of fire ignition methods and stove ventilation rates. Emission factors of PM2.5/PM10, CO, CO_2, NOx, SO_2 and H_2S were measured for bottom-lit updraft (BLUD) and top-lit updraft (TLUD) ignition methods and for three stove ventilation rates from informal braziers. The significance of the work lies in conducting experimental procedures that confirms no-cost or low-cost interventions to reduce exposure to household air pollution (HAP) as an interim measure until implementation of universal clean sustainable energy for all (se4all, 2015).

2 Experimental methods
2.1 Experimental stoves

Emissions were measured from coal fires set in a set of three representative artisan–manufactured braziers/*imbaulas*, purchased from users in residential areas of Johannesburg (Figure 1). Tests were conducted under laboratory conditions at the SeTAR Centre laboratory, University of Johannesburg.

Imbaula (brazier type) stoves are hand-made out of round metal drums with perforations of varying sizes around the sides, and a wire grate across the middle of the container to support the solid fuel. The *imbaulas* are found in three characteristic sizes,
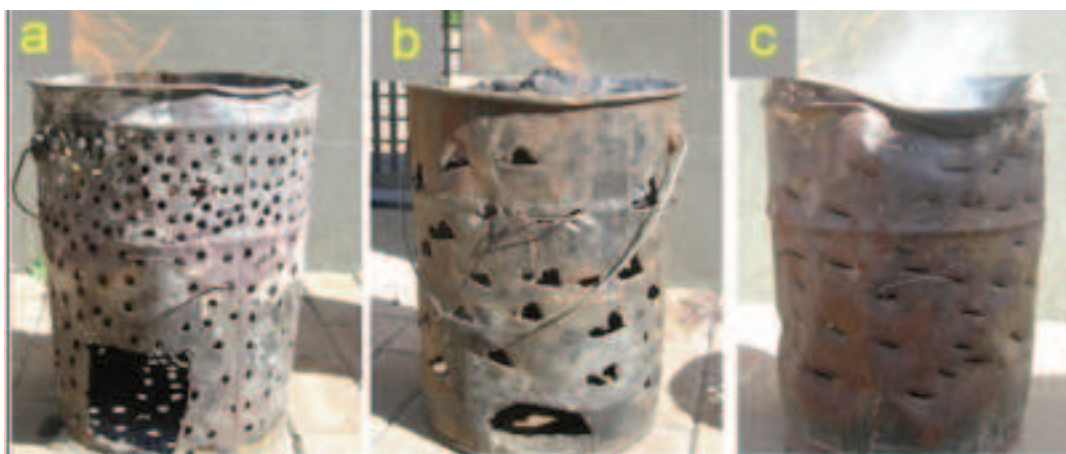
Figure 1: Field obtained braziers used in the experiments: (a) high ventilation; (b) medium ventilation; and (c) low ventilation

determined by three commonly available metal drums: 20 L metal paint drums- for domestic use; 70 L metal dustbins, or sectioned 200 L oil drums – for commercial purposes and typically used in street-side restaurants. In our experiments, only 20 L drums were used.

The devices for testing were selected with hole-sizes and densities to represent low, medium and high stove ventilation rates (taken as the total are of the holes). The fuel grate was kept at the same height for all three braziers. Further optimisation in terms of the location and density of holes below and above the grid was not considered at this stage of the investigation. The stove ventilation rates (hole areas) for the three field obtained braziers are given in Table 1.

Table 1: Area of holes for each for the field obtained cooking devices

Stove ventilation rate	Area of holes above the grate (cm²)	Area of holes below the grate (cm²)	Total hole area (cm²)
High ventilation	159	248	407
Medium ventilation	166	189	355
Low ventilation	91	63	154

2.2 Fuel analysis

Sufficiently large batches of coal were purchased to conduct a range of comparative tests, and were characterised for thermal content, elemental (proximate) analysis, moisture and ash content in third party laboratories using standard methods. The fuel samples were analysed on an air dry basis. Results for the proximate and ultimate analysis for the D-grade coal used in this study are presented in Table 1.

Prior to the experiments, the coal was crushed and sieved. Coal pieces in the range 20 mm<D<40 mm were used to minimise variability due to changes in air flow resistance in the packed bed resulting from non-uniform coal sizes. Prior to use, the fuel was stored in airtight containers. Each fuel batch was analysed for moisture content shortly before the commencement of each set of experiments.

2.3 Fire ignition methods

Two methods of igniting the coal fires were compared to the traditional bottom-lit updraft (BLUD) method, and the top-lit updraft (TLUD) method. The latter is colloquially known as the *Basa njengo Magogo* (BnM; it means 'to make fire like the old lady/grandmother'), and elsewhere as the Scotch fire ignition method. In the BLUD method the order of laying a fire is as follows: a few lumps of coal were added onto the grate, followed by paper, wood kindling, and then ignition, after which the bulk of the coal was added at an appropriate time after the wood fire is established. In our experiments, ~1 000 g of coal was placed onto a grate at the bottom of the brazier followed by 36 g of rolled paper and 360 g of pine wood chips. After ignition, about 2 000 g of coal was added on top of the already burning kindling (Makonese *et al*., 2014).

In the TLUD method, the order of laying the fire is reversed – first the bulk of coal was placed onto the fire grate followed by paper, and then wood kindling, with a few lumps of coal added on top of the already burning kindling at an appropriate time. As such, 2 000 g of coal was added to the bottom of the brazier onto the fuel grate followed by 36 g of paper and 360 g of kindling. After lighting the kindling, about 1 000 g of coal was added to the brazier above the kindling (Makonese *et al*., 2014).

2.4 Experimental set-up

An adaptation of the *hood* method (Ahuja *et al*., 1987) was used for evaluating emissions from coal-burning braziers. Since the experimental stoves did not have a flue, the stoves were placed under a collection hood attached to the dilution system which was responsible for the ducting and dilution of

Table 2: Proximate and ultimate analysis results of the D-grade coals used

| Parameter (Air Dry Basis)	Standard Method	Slater Coal D-Grade
Moisture content (%)	ISO 5925	3.5
Volatiles (%)	ISO 562	20.3
Ash (%)	ISO 1171	24.2
Fixed carbon (%)	By difference	52.0
Calorific value (MJkg^{-1})	ISO 1928	23.4
Calorific value (kcalkg^{-1})	ISO 1928	5590
Total sulphur (%)	ASTM D4239	0.63
Carbon (%)	ASTM D5373	62.6
Hydrogen (%)	ASTM D5373	2.72
Nitrogen (%)	ASTM D5373	1.43
Oxygen (%)	By difference	4.96
Total silica as SiO_2 (%)	ASTM D4326	58.6
Aluminium as Al_2O_3 (%)	ASTM D4326	27.6
Total iron as Fe_2O_3 (%)	ASTM D4326	6.63
Titanium as TiO_2 (%)	ASTM D4326	0.82
Phosphorous as P_2O_5 (%)	ASTM D4326	0.55
Calcium as CaO (%)	ASTM D4326	2.30
Magnesium as MgO (%)	ASTM D4326	0.83
Sodium as Na_2O (%)	ASTM D4326	0.42
Potassium as K_2O (%)	ASTM D4326	0.79
Sulphur as SO_3 (%)	ASTM D4326	1.10
Manganese as MnO_2 (%)	ASTM D4326	0.12
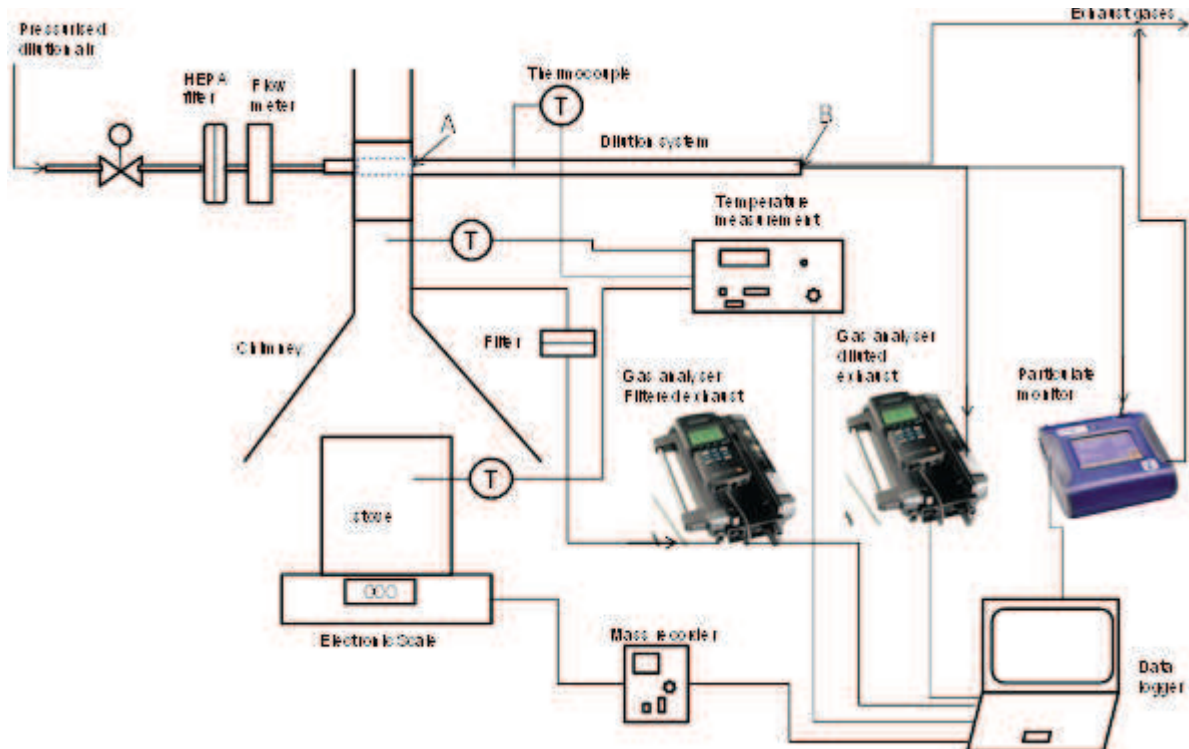
Figure 2: Schematic illustration of the experimental dilution setup for the SeTAR dilution system, showing the mixing point (A) and the sampling point (B) [Not drawn to scale]

exhaust emissions (Figure 2). Since a high forced extraction flowrate may influence the combustion characteristics of the stove (Bhattacharya *et al.*, 2002), an extractor fan was not used for drawing air through the hood and duct. The hood method can be used simultaneously with measurements for the determination of thermal parameters. This has the added advantage of enabling simultaneous meas-

urements of emissions and thermal parameters in a systematic and standard manner (Zhang *et al.*, 1999).

The sampling configuration for gases included, in sequence, a stainless steel tubing, a filter holder, and a flue gas analyser (Testo® 350XL/454) for the undiluted flue gas channel. For the diluted channel, the sampling configuration included, in sequence, the dilution system, Teflon tubing, and a second Testo® flue gas analyser. The Testo® measured CO_2, CO, NOx, H_2, H_2S, SO_2 and O_2. The monitor is equipped with standard measurement modules for O_2, CO, and NO_x. In addition to this, measurement modules for CxHy, NO_{low}, CO_{low}, SO_2, H_2S or CO_2 by infrared absorption are optionally available. The monitor uses electrochemical cells for H_2S and NOx gas measurements, whilst CO_2 is determined using a non-dispersive infrared cell and an oxygen balance is used for the calculation of excess air.

The sampling configuration for particulate matter included, in sequence, the SeTAR dilution system, Teflon tubing, and a particulate monitor (DustTrak DRX 8533 aerosol monitor). The DustTrak aerosol measures number–size fractions of the sampled aerosol by laser scattering. The instrument measures simultaneously in real time size-segregated fractions PM1, PM2.5, PM4, PM10, and Total Particle Mass –TPM, over a wide concentration range (0.001–150 mg m^{-3}).

2.5 Calculations

2.5.1 Determination of fuel moisture content
To determine the moisture content (MC) in the fuel, a small sample (∼ 50 g) of the coal was weighed on a calibrated scale with 0.1 g resolution and dried in an oven at 100º C for about 24 hours. The sample was taken out and weighed again a day after the initial 24 hrs had lapsed. To check that the coal fuel had attained dry mass, the exercise was repeated every 3 hours until a steady mass was attained. The percentage moisture content was calculated using the following equation:

$$MC_{wet} = \frac{MF_{wet} - MF_{dry}}{MF_{wet}} \times 100 \quad (1)$$

where MF_{wet} is the mass of the wet fuel and MF_{dry} is the mass of the dry fuel.

2.5.2 Determination of the dilution ratio
Carbon dioxide mixing ratios of the diluted and undiluted exhaust were measured to determine the dilution ratio. This eliminates the need for a venturi flow meter on the inlet, which has been shown to incur particle loss (Lipsky and Robinson, 2005; Hildeman *et al.*, 1989). For the purposes of this study, dilution ratio (DR) is defined by Equation 2:

$$DR = \frac{CO_{2_{meas\_undil}} - CO_{2_{amb}}}{CO_{2_{meas\_dil}} - CO_{2_{amb}}} \quad (2)$$

where CO_{2meas_undil} is the CO_2 measured from the undiluted exhaust, CO_{2meas_dil} is the CO_2 concentration measured from the diluted exhaust, and CO_{2amb} is the CO_2 concentration of the ambient environment. The CO_2 concentration of the undiluted exhaust is measured at the sample inlet whilst the diluted exhaust CO_2 concentration is monitored at the end of the tunnel after the sample is well mixed. Separate CO_2 monitors and sensors were used in the dilution channel to continuously monitor the dilution ratio of the system. Variations in the CO_2 concentration of the dilution air (CO_{2amb}) were negligible and did not impact the calculation of dilution levels based on Equation 2 (Lipsky & Robinson, 2005), which can thus be simplified to:

$$DR = \frac{CO_{2_{meas\_undil}}}{CO_{2_{meas\_dil}}} \quad (3)$$

2.5.3 Calculating the volume of emissions
Based on the chemistry of the burned fuel (raw fuel corrected for remaining materials) the stoichiometric volume of the combustion products was estimated. A gas sample was drawn from the emission stream and analysed for O_2, CO_2, CO, NOx, H_2, SO_2 and H_2S content using Testo gas analysers. The volumetric concentration of each gas is multiplied by λ (refers to the total air demand) to derive a total number of moles from which a total mass of each gas is calculated. The SeTAR data calculation sheet uses a chemical mass balance approach for the determination of Excess Air (EA). EA is the amount of air used in excess of the amount required for complete (stoichiometric) combustion of the fuel and is represented by Equation 4:

$$EA = \frac{O_{2_{meas}} - O_{2_{oxid}}}{O_{2_{det}} - (O_{2_{meas}} - O_{2_{oxid}})} \quad (4)$$

where 0_{2meas} is the measured O_2; $0_{2measoxid}$ is the O_2 required to complete the oxidation of incompletely burned gases and 0_{2det} is the total O_2 in all detected gases. The determination of the total volume of emissions from the fire is affected by the volume of diluting air in the sample. The calculation of the total air demand (λ) is made in the following manner:

$$λ = EA + 1 \quad (5)$$

This method reflected in Equations 4 and 5 can determine the total air demand λ even when a water gas shift reaction is taking place in the fire.

2.5.4 Determination of emission factors

The characteristics and quantities of gaseous and particulate emissions from residential coal combustion in typical braziers were determined and emission factors reported in an extensive experimental study. A large number of gaseous and particulate components were studied for the experimental fuels and devices with systematic variations in fuel characteristics (e.g. quality and moisture content), appliance and operational properties. Special focus was on detailed characterization of the emissions of CO, CO_2, NOx, H_2S, SO_2 and particulate matter. Statistical analyses were used to determine if there were significant differences in emission factors due to varying parameters. A two-tailed student T-test at the 95% confidence level was used for statistical evaluation of the thermal and PM data.

For the determination of gaseous emission factors, the SeTAR heterogeneous stove testing protocol relies on the carbon mass balance method, which has been previously employed for sampling open biomass combustion sources (Ward et al., 1996) and biofuels (Bertschi et al., 2003). This method relies on the ratio between pollutants and a fuel proxy (e.g. CO plus CO_2) in the exhaust gas to determine an emission factor (Roden et al., 2008; 2006). This approach requires a representative sample, so either multiple points in the plume must be sampled equally, or the entire plume must be collected and well mixed before sampling (Roden et al., 2008). For our experiments, exhaust gas samples from the braziers were extracted from the exit duct above the capture hood ~ 1.0 m above the fire into Teflon tubes. This approach allows for into a Teflon tube taken initial dilution through natural plume rise and entrainment.

For this study, the term emission factor is defined as concentration of a gas or particulate matter emitted by the stove, expressed in parts per million volume (ppmv), normalised to 0% excess air (oxygen). This provides the concentrations in undiluted air equivalent (i.e. sufficient air to provide stoichiometric combustion). For simplicity in representing the results of the mass balance of all emissions from complete and partial combustion of the fuel, we have chosen zero per cent excess air as a reference value for emission factor reporting. It is possible to convert emissions in ppmv to other units such as [g MJ^{-1} or mg MJ^{-1}] of fuel. First, there is a need to determine the net heat gained (H_{NET}) from the fuel. This variable is the heat retained by a cooking vessel during a burn cycle and is expressed in units of Mega Joules [MJ]. It includes the heating of the pot and its contents plus the heat of evaporation of water, but excludes other heat flows through the pot, specifically radiative and convective losses from the pot sides and top.

The mass of detected PM2.5 or PM10 is first multiplied by any dilution factor applied by the equipment, then multiplied by □ to derive the total mass emitted. This method is able to track and correctly determine the performance of the stove in real time while burning fuels in an inhomogeneous manner, as is often the case with biomass and coal fuels. At a minimum, the standard reporting metrics for the particle mass concentration include mass of PM emitted per net MJ of energy delivered into the pot, or mass of PM emitted per net MJ of energy delivered from the fire. For example, the mass of CO_2, CO, PM2.5 and PM10 in grams; NOx, H_2S, and SO_2 in milligrams emitted during a burn cycle are determined and divided by the net heat gained H_{NET}, yielding emission factors with units of mass of emissions per net MJ [g MJ^{-1} or mg MJ^{-1}]. For example:

$$CO_{EF} = \frac{CO[g]}{H_{NET}[MJ]} \tag{5}$$

$$PM10_{EF} = \frac{PM10[g]}{H_{NET}[MJ]} \tag{6}$$

$$H_2S_{EF} = \frac{H_2S[mg]}{H_{NET}[MJ]} \tag{7}$$

2.5.5 Statistical analysis

An F-Test for the null hypothesis that two normal populations have the same variance was used in our data analyses. This test can be used in practice, with care, particularly where a quick check is required, and subject to associated diagnostic checking. We used large alpha levels (at least 0.05) and balanced layouts as the F-Test is relatively robust. The F-Test is used to determine the type of T-test to use (i.e. T-test: Two sample assuming equal variances or T-test: Two sample assuming unequal variances). A p-value >0.05 indicates the compared values are the same, while a p-value <0.05 indicates that the compared values differ with a 95% confidence.

The two-tailed student T-test at the 95% confidence level was applied for statistical evaluation of the emissions results from tests of the two ignition methods, and of the three stove ventilation rates.

3. Results and discussion
3.1 Effect of ignition method

Gas emission factors for the field-designed stoves, using D-grade coal, for top-lit and bottom-lit methods are presented in Table 3, together with a statistical comparison of the differences. The emission factors are based on integration over the combustion cycle, from ignition until three hours had elapsed, or fuel burnout occurred, whichever was the soonest.

When comparing the TLUD and BLUD ignition methods, there are no statistically significant differences (p>0.05) in CO emission factors at high and medium ventilation rates. However, there was found to be a significant difference (p<0.05) in CO emission factors between the fire-ignition methods at low ventilation rates (Table 3) – there was a 20% increase in CO when using the TLUD method. This result is rather counterintuitive – it is expected that CO emissions will increase with poor stove ventilation rates (typical of BLUD fires). In the absence of sufficient oxygen, CO formed by oxysorption on the surface of the char is unable to further oxidise to CO_2 and as result high CO emissions are detected in the flue gas. Thus, a lack of oxygen results in smouldering combustion conditions which favour the emission of products of incomplete combustion, including CO. Further detailed explanations need to be sought to explain this anomaly.

CO_2 emissions do not show significant differences with a change in the ignition method. This is because CO_2 emissions are dependent on the carbon content of the coal, and since the fuel was taken from the same batch, little to no change in total CO_2 emissions is expected. CO is influenced by poor combustion conditions during initial phases of pyrolization, but the majority of CO arises during coking combustion (Figure 3).

Emission factors of NOx, SO_2 and H_2S show statistically significant differences between ignition methods. The use of the TLUD method resulted in a 13% decrease in NOx emissions and an average 65% and 90% reduction in SO_2 and H_2S emissions, respectively (Table 3). SO_2 emissions from the BLUD method were three-fold higher compared to the TLUD method, whereas H_2S EFs were ten-fold higher in the BLUD compared to the TLUD

method. This indicates that the TLUD ignition method has potential to reduce the amount of SO_2 and H_2S emitted to the atmosphere during residential coal combustion. Different results to the ones reported in this study on the dependence of H_2S and SO_2 emission on the fire-ignition method have been reported in the literature. Le Roux *et al*. (2009) found that the ignition method does not have an influence on SO_2 emissions.

Our results for NOx, SO_2 and H_2S emissions are counterintuitive, for which further explanation has to be sought. It is expected that NOx, SO_2 and H_2S emissions do not vary significantly with the ignition method, when other parameters are held constant. Total sulphur emissions (SO_2 + H_2S) are a constant that is dependent on the sulphur content of the coal and, based on mass conservation, ought not to vary significantly if the coal is taken from the same batch and combusted to completion. However, the ratio of SO_2 and H_2S could vary, depending on combustion conditions. H_2S formation is enhanced in the restricted oxygen environment of the BLUD ignition method (i.e. with the rising column of hot gases including N_2, CO_2, H_2O, H_2 and CO during coal pyrolysis, sulphur competes with hydrogen for available oxygen, and may combine with H^+ to form H_2S). The resultant total emission factors should indicate an enhancement of H_2S with a corresponding decrease in SO_2 emissions, and vice versa for the respective ignition methods. However, results presented here show that the sum of H_2S and SO_2 gases do not show S mass conservation, indicating possible systematic errors in the analysis or processing of obtained data. The total S recovered is 2% to 6% of the S as analysed in the fuel (See Section 2.2 for results on fuel analysis) for the BLUD and the TLUD ignition methods, respective-

Table 3: Gas emission factors for the field- stoves using D-grade, for the TLUD and the BLUD ignition methods

| Pollutant	Ventilation rate	BLUD Emission factor ± StDev [g MJ⁻¹]		TLUD Emission factor ± StDev [g MJ⁻¹]		Statistical analysis % diff. between BLUD and TLUD	T-Stat	P-Value	Sig @ 95%
CO	High	4.1	0.3	4.0	0.2	-3%	1	0.58	No
	Medium	4.2	0.4	4.1	0.4	-4%	1	0.61	No
	Low	4.6	0.3	5.5	0.2	20%	-4	0.01	Yes
CO_2	High	102	4	100	6	-2%	1	0.61	No
	Medium	102	5	99	3	-3%	1	0.53	No
	Low	98	6	101	5	2%	0	0.72	No
NO_x	High	195	8	168	10	-14%	4	0.02	Yes
	Medium	188	10	163	4	-13%	4	0.02	Yes
	Low	187	11	161	9	-14%	3	0.04	Yes
SO_2	High	28	2	9	0.5	-68%	19	0.00	Yes
	Medium	31	1	11	0.3	-66%	23	0.00	Yes
	Low	27	2	10	0.8	-63%	14	0.00	Yes
H_2S	High	1.7	0.2	0.19	0.02	-89%	17	0.00	Yes
	Medium	1.6	0.1	0.17	0.02	-89%	23	0.00	Yes
	Low	2.0	0.2	0.16	0.01	-92%	16	0.00	Yes
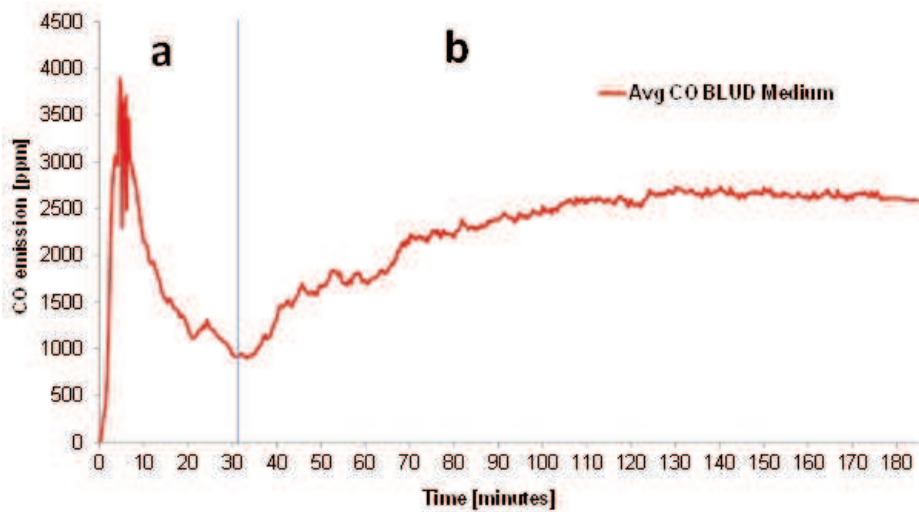
Figure 3: Typical CO [ppm] time series plot from a BLUD ignition method, for the medium ventilation case: (a) ignition of kindling and pyrolysis of the coal; and (b) coking phase

ly. Based on principles of conservation of mass, the total S recovered from the analysis should be similar to the S as analysed in the elemental content of the fuel. There is a need to analyse the remaining ash for S content to see how much of it remained in the ash especially for TLUD fires. A reason for this anomaly could be that no attempt was made in this study to homogenise the received fuel samples (about 1 000 kg of fuel) and this could have resulted in significant variations in the resultant emission factors. In future evaluations, one could first homogenise the received fuel sample and store it under a dry nitrogen atmosphere.

During the combustion experiments, we noticed that emissions of SO_2 and H_2S were released early in the combustion cycle, with a single peak emission rate for each gas during the first hour of combustion. This is in contrast to CO and PM emissions that showed were released in varying amounts

throughout the burn cycle, Figure 4 shows an emissions time graph of H_2S and SO_2–indicating that H_2S forms during the pyrolization phase and is low during coking phase. During the coking phase, when sufficient O_2 is available, SO_2 dominates the low residual S emissions. The emission of these pollutants decreased gradually after their emission peaks, indicating H_2S and SO_2 may be the dominant precursors/catalysts for the formation of organic condensation particles during the ignition phase (Wang *et al.*, 2012).

Particle emission factors for the laboratory designed stoves, using D-grade coal, for TLUD and BLUD ignition methods are presented in Table 4, together with a statistical evaluation of the differences. The emission factors, expressed as particulate mass per MJ of energy in the combusted fuel, are based on integration over the combustion cycle from ignition until three hours had elapsed, or fuel
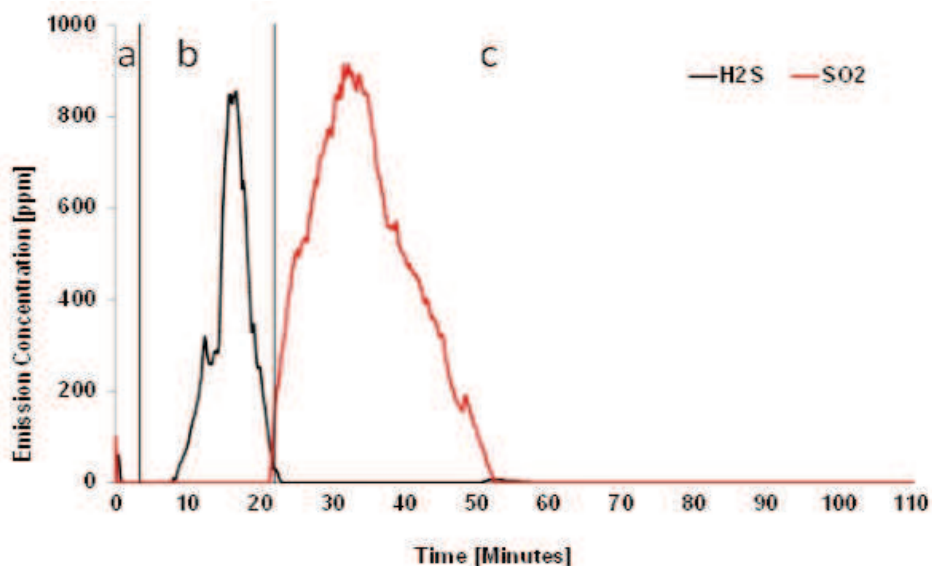

Figure 4: Typical SO₂ and H₂S emissions [ppm], from a BLUD fire (low ventilation rate, field stove): Phase (a) ignition of the kindling; (b) – pyrolysis of the coal; and phase (c) – coking

Table 4: Particle emission factors for the field-designed coal braziers using D-grade coal in both the BLUD and the TLUD fires

Pollutant	Ventilation rate	BLUD method Emission factor ± StDev [g MJ^{-1}]		TLUD method Emission factor ± StDev [g MJ^{-1}]		Statistical analysis % diff. between BLUD and TLUD	T-Stat	P-Value	Sig @ 95%
PM2.5	High	1.3	0.1	0.2	0.02	-81%	15	0.0001	Yes
	Medium	2.9	0.3	0.6	0.05	-78%	15	0.0001	Yes
	Low	3.3	0.2	0.7	0.06	-80%	23	0.0000	Yes
PM10	High	1.3	0.1	0.2	0.02	-81%	15	0.0001	Yes
	Medium	2.9	0.2	0.6	0.05	-78%	15	0.0001	Yes
	Low	3.3	0.2	0.7	0.06	-80%	23	0.0000	Yes

* PM10 in this report is taken as the sum of all mass less than 10 μm, including the PM2.5 fraction

burnout, whichever was the soonest.

From Table 4, it can be seen that PM2.5 emissions constitute more than 90% of PM10 emissions. There are statistically significant differences (p<0.05) in PM emission factors between the fire-ignition methods. The top-lit ignition method reduces PM10/PM2.5 by 76% to 80% compared to the BLUD method. This result is similar to assertions made by Le Roux *et al*. (2009) who found a reduction in particulate emissions of between 78% and 92% when using the TLUD ignition method. However, Le Roux *et al*. (2009) do not specify ventilation rate of their single tested device, nor on coal moisture and coal lump size.

3.2 Effects of ventilation rates
The effect of ventilation rates on the emissions of CO, CO$_2$, SO$_2$ and H$_2$S were investigated. Three stoves with different ventilation rates were used for each ignition method, with moisture content and other parameters held constant. The results of these experiments for three field-acquired braziers are presented in Table 5.

When comparing the high and medium ventilation rates, results show that there is a significant difference (p<0.05) in CO emissions, but not for CO$_2$, SO$_2$ and H$_2$S emissions. When comparing medium and low ventilation rates, there are no significant differences (p>0.05) between the respective emissions of CO, CO$_2$, SO$_2$ and H$_2$S. When comparing the high and low ventilation rates, results show that there is a significant difference (p<0.05) in CO emissions, but not for CO$_2$, SO$_2$ and H$_2$S emissions. This shows that ventilation rates of the tested devices influence CO emissions, but may not affect the combustion characteristics of CO$_2$, SO$_2$ and H$_2$S.

The effect of ventilation rates on the emissions of PM10 is investigated. Three stoves with different ventilation rates were evaluated for each ignition method, with moisture content and other parameters held constant. The results of the experiments for the lab-designed braziers are presented in Table 6.

When comparing high and medium stove ventilation rates, results show that there are significant differences (p<0.05) in PM10 emissions for both ignition methods. However, when comparing medium and low ventilation rates, results show that there are no significant differences (p>0.05) in PM10 emissions for either ignition method. For the high and low stove ventilation rates, results show that there are significant differences (p<0.05) in PM10 emissions for both ignition methods. The trends are that the more air holes there are in a brazier, the less the particulate emissions. A change from low to high stove ventilation rates result in a 50% reduction of particulate emissions. However, the reduction due to increased ventilation is not as great as the reduction due the switch to the top-lit ignition method. A well ventilated brazier when used in conjunction with the TLUD ignition method has the potential to have the maximum reduction of PM emissions from coal fires in informal braziers. It should be noted that the range of ventilation hole areas as defined in this study, were based on representative samples collected from field observations and as such, these potential reductions can be applied in estimating benefits that could be obtained in implementation campaigns.

The performance of the braziers can be improved by increasing ventilation through the fuel bed. An increase in the oxygen content allows for improved combustion conditions and increased combustion temperatures that can lead to improved heat transfer. If excess air is limited, inefficient combustion results with consequent increased emissions of products of incomplete combustion including soot (black carbon), smoke, and carbon monoxide. Conversely, relatively high excess air can lower the combustion efficiency–excessive airflow rates/ high ventilation rates may increase the convective cooling of the fire, thereby affecting the mixing and local concentration of fuel/oxidant in the gas stream. In addition, air velocity rates may directly affect the reaction rate of the condensed phase. As such, a balance should be found between providing the optimal amount of excess air. The most important issue in combustion research is the improvement of combustion efficiency to minimise the emission of harmful pollutants. Increased ventilation in a stove

Table 5: Emission factors of CO, CO_2, H_2S and SO_2 as a function of stove ventilation rates

Ignition method	Ventilation rate	CO emission factor [g MJ⁻¹]	T-start	P-value	Sig @ 95%	CO_2 emission factor [g MJ⁻¹]	T-start	P-value	Sig @ 95%
BLUD	High	4.5 ± 0.1	-6.6	0.0	Yes	98 ± 6	0.2	0.9	No
	Medium	6.2 ± 0.3				98 ± 6			
	Medium	6.2 ± 0.3	-0.5	0.7	No	98 ± 6	0.1	0.9	No
	Low	6.4 ± 0.5				97 ± 5			
	High	4.5 ± 0.1	-5.5	0.0	Yes	98 ± 6	0.4	0.7	No
	Low	6.4 ± 0.5				97 ± 5			
TLUD	High	4.6 ± 0.2	-1.1	0.4	No	99 ± 2	1.0	0.4	No
	Medium	5.0 ± 0.4				96 ± 2			
	Medium	5.0 ± 0.4	-2.3	0.2	No	96 ± 2	-2.1	0.2	No
	Low	5.7 ± 0.2				102 ± 3			
	High	4.6 ± 0.2	-5.6	0.0	Yes	99 ± 2	-1.2	0.4	No
	Low	5.7 ± 0.2				102 ± 3			

Ignition method	Ventilation rate	SO_2 emission factor [g MJ⁻¹]	T-start	P-value	Sig @ 95%	H_2S emission factor [g MJ⁻¹]	T-start	P-value	Sig @ 95%
BLUD	High	30.7 ± 1.1	1.9	0.2	No	1.4 ± 0.1	-1.1	0.4	No
	Medium	29.1 ± 0.3				1.6 ± 0.2			
	Medium	29.1 ± 0.3	3.9	0.1	No	1.6 ± 0.2	-1.1	0.4	No
	Low	26.0 ± 1.1				1.7 ± 0.7			
	High	30.7 ± 1.1	4.3	0.1	No	1.4 ± 0.1	-2.9	0.1	No
	Low	26.0 ± 1.1				1.7 ± 0.7			
TLUD	High	10.1 ± 0.2	1.1	0.4	No	0.15 ± 0.01	-1.3	0.3	No
	Medium	9.7 ± 0.3				0.16 ± 0.01			
	Medium	9.7 ± 0.3	1.1	0.4	No	0.16 ± 0.01	-1.3	0.3	No
	Low	9.4 ± 0.3				0.18 ± 0.01			
	High	10.1 ± 0.2	2.2	0.2	No	0.15 ± 0.01	-2.4	0.1	No
	Low	9.4 ± 0.3				0.18 ± 0.01			

Table 6: PM10 emission factors from lab-designed braziers at different stove ventilation rates

Ignition method	Ventilation rate	PM10 emission factor [g MJ⁻¹]	T-start	P-value	Sig @ 95%
BLUD	High	1.3 ± 0.1	-5	0.03	Yes
	Medium	1.9 ± 0.1			
	Medium	1.9 ± 0.1	-4	0.06	No
	Low	2.5 ± 0.2			
	High	1.3 ± 0.1	5	0.03	Yes
	Low	2.5 ± 0.2			
TLUD	High	0.3 ± 0.02	-6	0.02	Yes
	Medium	0.4 ± 0.03			
	Medium	0.4 ± 0.03	-4	0.07	No
	Low	0.6 ± 0.06			
	High	0.3 ± 0.02	-8	0.02	Yes
	Low	0.6 ± 0.06			

may also be achieved by using forced draft/ ventilation, as in some wood pellet and biomass burning gasifiers. However, this option is not viable on brazier stoves. The target market for these stoves remains the low-income (energy poor) households situated on the Highveld region of South Africa.

The significant effect of stove ventilation rates and fire-ignition methods on the gaseous and condensed matter (smoke) emission factors observed in this study highlights the vital role of these parameters in the performance evaluation. For future tests and performance evaluation exercises, testing protocols should specify or restrict stove ventilation rates and ignition methods to be used, to avoid any bias that may emanate from different stove ventilation rates and fire-ignition methods. For example, prediction models are used to estimate the amount of pollutant emissions in a given geographical location, and the estimates are based on emission factors from actual stove tests. As such, it is recom-

mended that stove ventilation rates and the ignition methods employed should be evaluated through extensive field surveys in the estimation of pollutant emission factors to be used in the models.

4 Conclusion

In this study, the impacts of fire ignition methods and ventilation rates on the emissions of gaseous and condensed matter (smoke) particles were investigated. Measured gaseous and PM emission factors were significantly different with changes in the ignition method and stove ventilation rates. Emissions of gaseous and particulate matter from incomplete combustion can be minimised by design optimisation of the braziers. Ideally, the emissions of condensed matter particles (which form the bulk of emitted particles in residential coal fires) can be reduced by allowing for good mixing between the volatile gas from the fuel bed and the air supply and by ensuring a long residence time in the high temperature oxygen rich post combustor zone to allow for complete oxidation. Thus a brazier/*imbaula* with high ventilation rates has a potential to reduce emissions of CO and PM. However, results presented herein provide useful information on the important role of ignition method and ventilation rates in the performance evaluation of cookstoves, and how the emission factors obtained from these evaluations may be used in accurate prediction of total emissions.

Further studies are needed to understand the influences of other combustion parameters on emission performance of fixed-bed residential coal burning devices. Reported low, medium and high ventilation rates were based on the empirical observations of stoves in use. Future improvements could explore additional factors that include varying the size of ventilation holes, and location density above or below the fuel bed, and the median size and size distribution of the fuel. We kept the fuel median size and size distribution constant between 20 mm and 40 mm throughout the course of this study. There is a need to investigate the effect of hole density pattern on ventilation rates by clustering primary air holes below the fuel grate and a limited number of secondary air holes above the level of the packed fuel bed. Repeats of experiments are needed to determine whether anomalies found in this study are reproducible or not and to determine the range of emission factors when repeating experiments under the same operating conditions.

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